$$CHESS \& LEPP$$

$$\frac{d}{d\vartheta}J_{x} = \cos(\widetilde{\psi}_{x} + \varphi_{x})\sqrt{2J_{x}\beta_{x}}\Delta f_{x}\frac{L}{2\pi} \quad , \quad \frac{d}{d\vartheta}\varphi_{x} = \upsilon_{x} - \sin(\widetilde{\psi}_{x} + \varphi_{x})\sqrt{\frac{\beta_{x}}{2J_{x}}}\Delta f_{x}\frac{L}{2\pi}$$

$$\frac{d}{d\vartheta}J_{y} = \cos(\widetilde{\psi}_{y} + \varphi_{y})\sqrt{2J_{y}\beta_{y}}\Delta f_{y}\frac{L}{2\pi} \quad , \quad \frac{d}{d\vartheta}\varphi_{y} = \upsilon_{y} - \sin(\widetilde{\psi}_{y} + \varphi_{y})\sqrt{\frac{\beta_{y}}{2J_{y}}}\Delta f_{y}\frac{L}{2\pi}$$

$$\frac{d}{d\vartheta}\vec{\varphi} = \vec{\partial}_{J}H \quad , \quad \frac{d}{d\vartheta}\vec{J} = -\vec{\partial}_{\varphi}H \quad , \quad H(\vec{\varphi},\vec{J},\vartheta) = \vec{\upsilon}\cdot\vec{J} - \frac{L}{2\pi}\int_{0}^{\vec{x}}\Delta\vec{f}(\hat{\vec{x}},s)d\hat{\vec{x}}$$

Counling Resonances

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The integral form can be chosen since it is path independent. This is due to the Hamiltonian nature of the force: $\Delta f_{x,y}(x, y, s) = -\partial_{x,y}\Delta H(x, y, s)$

Single Resonance model for two dimensions means retaining only the amplitude dependent tune shift and one term in the two dimensional Fourier expansion:

$$H(\vec{\varphi}, \vec{J}, \vartheta) = \vec{v} \cdot \vec{J} + H_{00}(\vec{J}) + H_{n\vec{m}}(\vec{J})\cos(n\vartheta + m_x\varphi_x + m_y\varphi_y + \Psi_{n\vec{m}}(\vec{J}))$$

For $n + m_x\upsilon_x + m_y\upsilon_y \approx 0$
 $m_x\varphi_x + m_y\varphi_y = \vec{m} \cdot \vec{\varphi}$





 $n + m_x v_x + m_y v_y \approx 0$ means that oscillations in y can drive oscillations in x in $x'' = -K x + \Delta f_x(x, y, s)$

The resonance term in the Hamiltonian then changes only slowly:

$$\begin{split} H(\vec{\varphi}, \vec{J}, \vartheta) &= \vec{v} \cdot \vec{J} + H_{00}(\vec{J}) + H_{n\vec{m}}(\vec{J}) \cos(n\vartheta + \vec{m} \cdot \vec{\varphi} + \Psi_{n\vec{m}}(\vec{J})) \\ \frac{d}{d\vartheta} \vec{\varphi} &= \vec{\partial}_J H \quad , \quad \frac{d}{d\vartheta} \vec{J} = -\vec{\partial}_{\varphi} H \\ J &= \vec{m} \cdot \vec{J} \\ J_{\perp} &= m_x J_x - m_y J_y = \vec{m} \times \vec{J} \quad \Rightarrow \quad \frac{d}{d\vartheta} J_{\perp} = 0 \\ \end{split}$$

 $n+|m_x|v_x-|m_y|v_y\approx 0 \Rightarrow |m_x|J_x+|m_y|J_y=const.$

Sum resonances lead to unstable motion since:

$$n+|m_x|v_x+|m_y|v_y\approx 0 \Rightarrow |m_x|J_x-|m_y|J_y=const.$$



Resonances Diagram



$n + m_x v_x + m_y v_y \approx 0$ means that oscillations in y can drive oscillations in x in



$$x'' = -K x + \Delta f_x(x, y, s)$$

All these resonances have to be avoided by their respective resonance width.

The position of an accelerator in the tune plane s called its Working Point.