

Recap: Harmonic Motion

• Simple Harmonic Motion:

$$\rightarrow x(t) = x_{\max} \cos(\omega t + \phi) \Leftrightarrow v(t) = -v_{\max} \sin(\omega t + \phi)$$

$$\Leftrightarrow a(t) = -a_{\max} \cos(\omega t + \phi) = -\omega^2 x(t)$$

$$a_{\max} = \omega^2 x_{\max} = \omega v_{\max}$$

$$v_{\max} = \omega x_{\max}$$

$$\rightarrow \text{acceleration } a(t) = -\omega^2 x(t) \Rightarrow \text{Force} = F_{\text{net}} = -m\omega^2 x$$

linear restoring force

$$\rightarrow \text{Mass on spring: } \omega_0 = \sqrt{\frac{k}{m}}$$



$$\rightarrow \text{Simple pendulum: } \omega_0 = \sqrt{\frac{g}{L}}$$



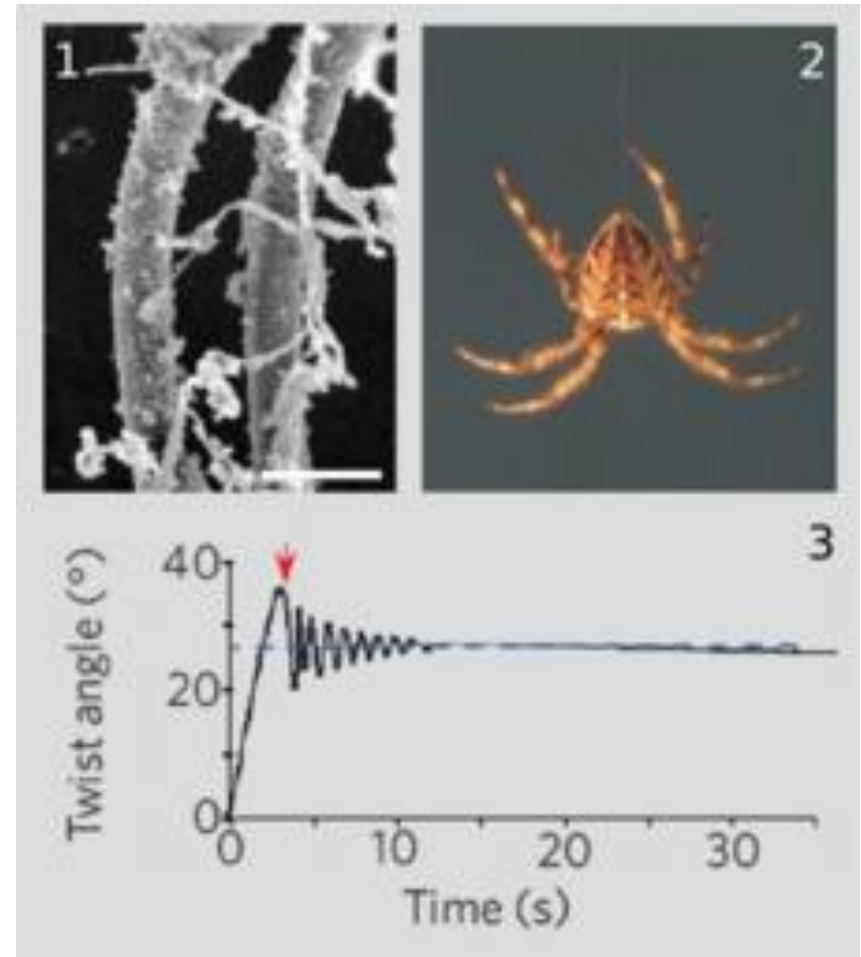
} $\omega_0 =$ "natural" angular frequency of oscillation

Today:

- Damped Oscillations (not damned oscillations...)
- Driven Oscillations

Twist angle of spider suspended from its silk thread

⇒ strong damping (that's why they hardly ever twist!)

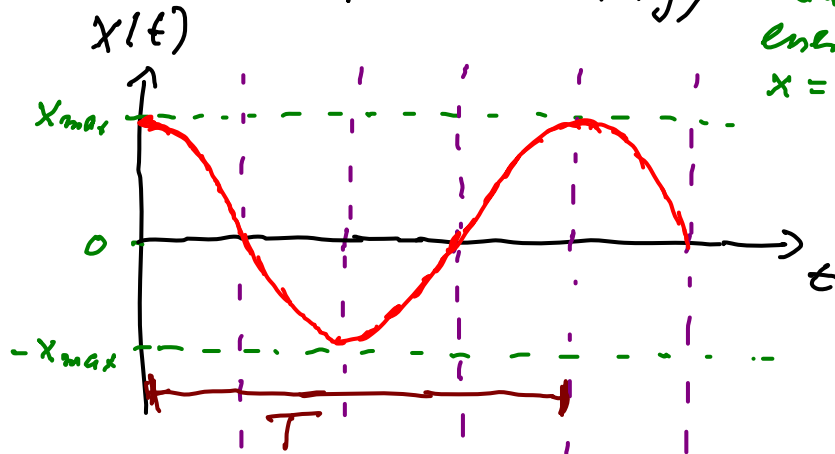


For SHM: \leftarrow kinetic energy

$$E_{\text{mech}} = \underbrace{K(t)}_{\text{kinetic energy}} + \underbrace{U_{\text{sp}}(t)}_{\text{potential energy}} = \frac{1}{2} m v_{\text{max}}^2 = \frac{1}{2} k x_{\text{max}}^2 = \text{const}$$

maximum kinetic energy, when $x=0$, and $U=0$

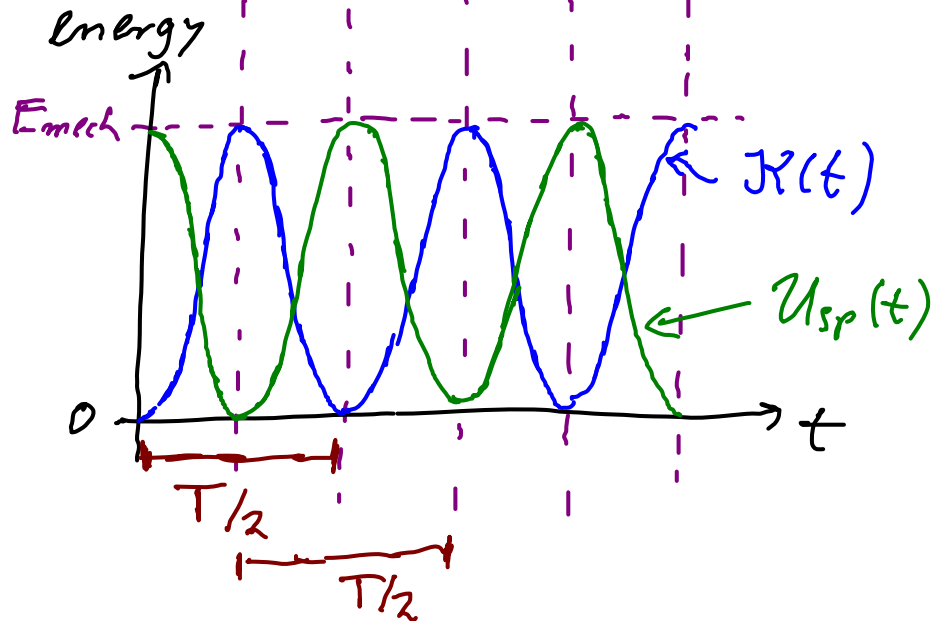
spring const.
maximum potential energy, when $x = \pm x_{\text{max}}$ and $K=0$



Notes:

- $K(t)$ and $U(t)$ are 180° out of phase, so that $K(t) + U(t) = \text{const}$

- $x(t)$ oscillates at $\omega = 2\pi/T$
 K, U oscillate at $2\omega = 2\pi/T/2$

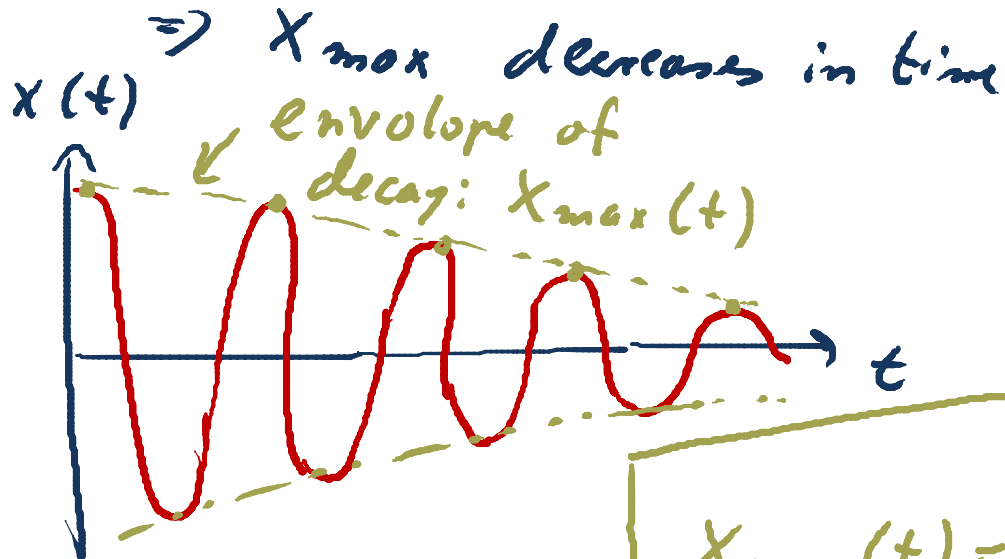


→ Realistic Case: Damped Oscillations

- with friction $\neq 0 \Rightarrow W_{\text{non-cons. forces}} \neq 0$

$\Rightarrow E_{\text{mech}}$ decreases in time

$\Rightarrow X_{\text{max}}$ decreases in time



$$x(t) = X_{\text{max}}(t) \cos(\omega t + \phi)$$

in many cases:

$$X_{\text{max}}(t) = X_{\text{max}}(t=0) e^{-t/(2\tau)}$$

decays exponentially
in time

$X_{\text{max},0}$

$\tau = \text{constant}$

$[\tau] = \text{sec}$

⇒ for energy:

$$\begin{aligned} E_{\text{mech}}(t) &= K(t) + U(t) \approx U_{\text{max}}(t) = \frac{1}{2} k x_{\text{max}}^2(t) \\ &= \frac{1}{2} k x_{\text{max}}(t=0)^2 \left(e^{-t/2\tau} \right)^2 \\ &= E_{\text{mech}}(t=0) \cdot e^{-t/\tau} \end{aligned}$$

$$\Rightarrow E_{\text{mech}}(t) = E_{\text{mech}}(t=0) e^{-t/\tau}$$

τ = energy decay time = time for energy to decay by a factor $1/e = e^{-1} = 0.37$

stronger damping ⇒ τ decreases ⇒ faster decay

⇒ after time interval $\Delta t = \tau$

$$\begin{aligned} E_{\text{mech}}(t + \tau) &= E_{\text{mech}}(t=0) e^{-(t+\tau)/\tau} \\ &= \underbrace{E_{\text{mech}}(t=0) e^{-t/\tau}}_{E_{\text{mech}}(t)} \cdot \underbrace{e^{-\tau/\tau}}_{1/e} = 0.37 \cdot E_{\text{mech}}(t) \end{aligned}$$

⇒ in each time interval $\Delta t = \tau$, the mechanical energy decays by a multiplicative factor $1/e = 0.37$.

$$\begin{array}{ccc}
 t=0 & t=\tau & t=2\tau \\
 E = E_0 & \xrightarrow{\Delta t = \tau} E(\tau) = \frac{1}{e} E_0 & \xrightarrow{\Delta t = \tau} E(2\tau) = \frac{1}{e} \left(\frac{1}{e} E_0 \right) = \frac{E_0}{e^2}
 \end{array}$$

$$\begin{array}{ccc}
 X_{\max} = X_{\max,0} & X_{\max}(\tau) = \frac{X_{\max,0}}{\sqrt{e}} & X_{\max}(2\tau) = \frac{1}{\sqrt{e}} \left(\frac{X_{\max,0}}{\sqrt{e}} \right) \\
 \uparrow & \uparrow & = \frac{X_{\max,0}}{e} \\
 X_{\max}(t) = X_{\max,0} e^{-t/2\tau} & \uparrow e^{-1/2} = \frac{1}{\sqrt{e}} &
 \end{array}$$

equal time intervals \Rightarrow equal multiplicative factors e
 in E_{\max} , X_{\max} , V_{\max} ...

similar:

$$\begin{array}{ccc}
 t=0 & t=10_s & t=20_s = 2 \cdot 10_s \\
 E_0 & \xrightarrow{\Delta t = 10_s} E(10_s) = \frac{1}{c} E_0 & \xrightarrow{\Delta t = 10_s} E(20_s) = \frac{1}{c} \left(\frac{E_0}{c} \right) \\
 & & \text{same factor} \nearrow \\
 & & \frac{1}{c} \text{ every } \Delta t = 10_s = \frac{E_0}{c^2}
 \end{array}$$

The mechanical energy of a damped harmonic oscillator decays by a **factor of 4 in 10 s.**

$$E_0 \rightarrow \frac{1}{4} E_0 \text{ in } \Delta t = 10 \text{ s}$$

How long does it take to decay by a **factor of 16?**

exponential decay:

$$E_{\text{mech}}(t=10\text{s}) = E_{\text{mech}}(t=0) e^{-10\text{s}/\tau} = \frac{1}{4} E(t=0)$$

$$\begin{array}{ccc} t=0 & t=10\text{s} & t=20\text{s} \\ E(0) & \rightarrow E = \frac{E_0}{4} & \rightarrow E = \frac{1}{4} \left(\frac{E_0}{4} \right) = \frac{E_0}{16} \end{array}$$

$$\begin{aligned} E_{\text{mech}}(t=20\text{s}) &= E(t=0) e^{-20\text{s}/\tau} \\ &= E(0) \underbrace{e^{-10\text{s}/\tau}}_{1/4} \cdot \underbrace{e^{-10\text{s}/\tau}}_{1/4} \end{aligned}$$

A. 10 s

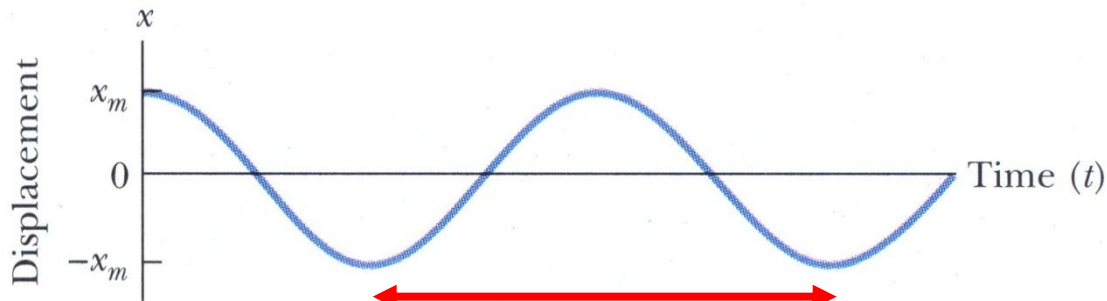
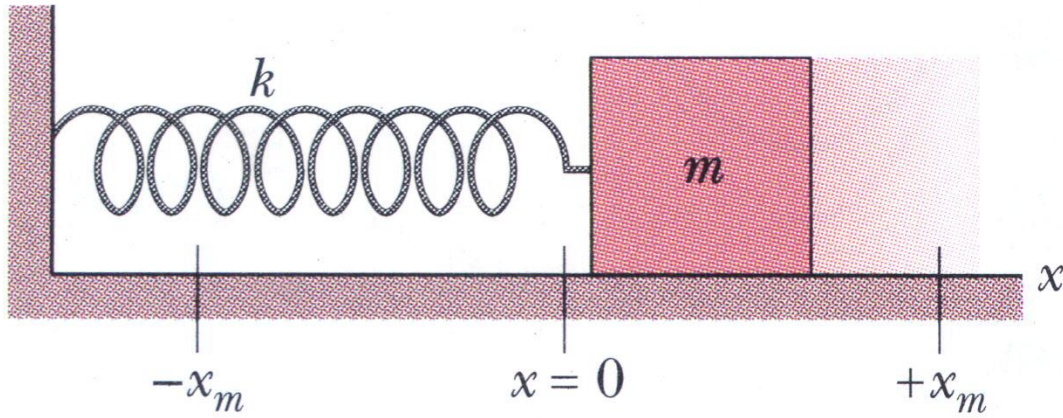
B. 20 s

C. 40 s

D. 80 s

E. 160 s

Harmonic Oscillator:



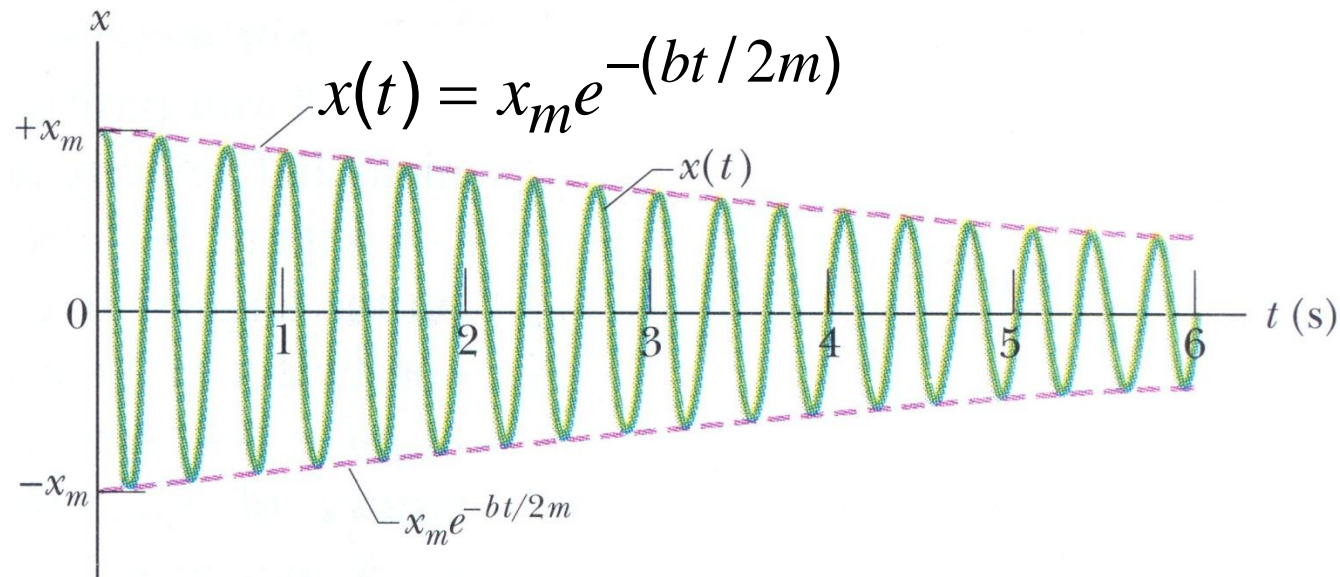
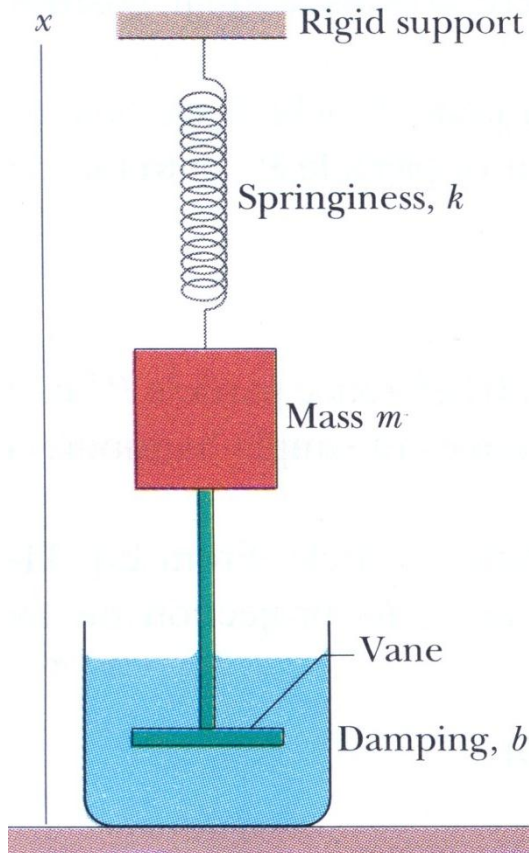
$$T = 2\pi / \omega_0$$

$$\omega_0 = \sqrt{\frac{k}{m}}$$

*natural
angular frequ.
without friction*

Damped Harmonic Oscillator:

Damping force: $F = -bv$



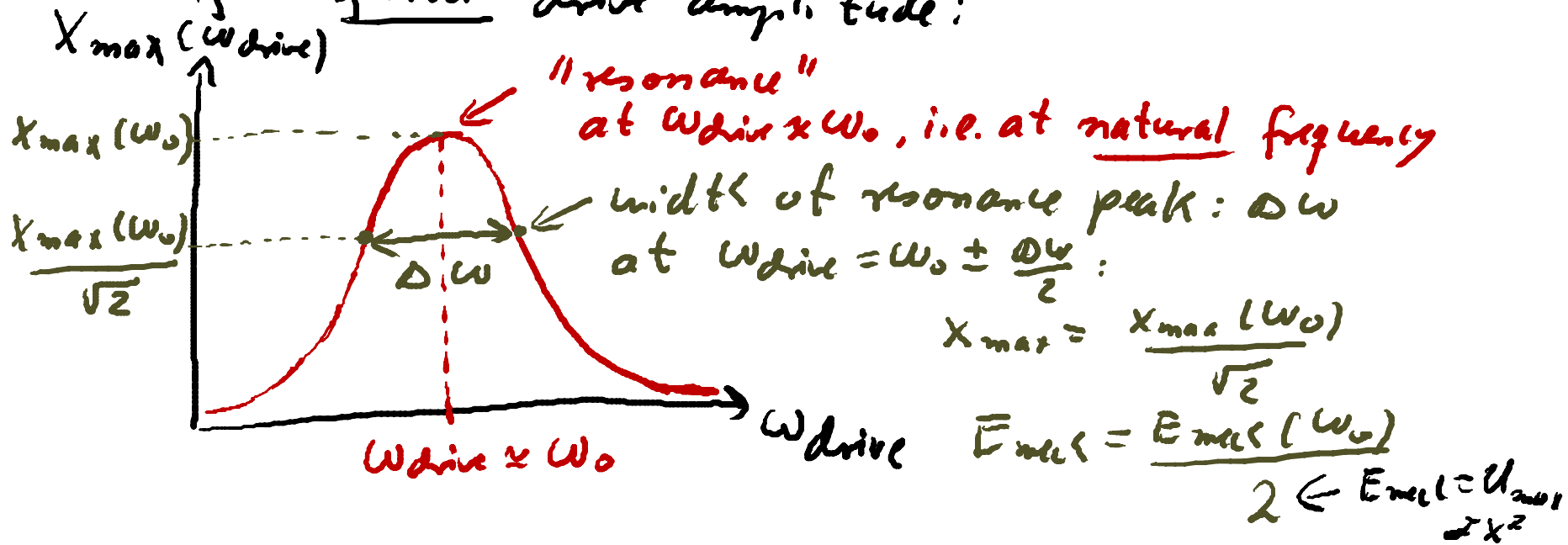
$$\omega' = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}} \approx \sqrt{\frac{k}{m}} = \omega_0$$

Energy decay time $\tau = \frac{m}{b}$

- Forced (driven) oscillations and resonance:
 - drive a damped oscillator continuously at frequency $\omega_{drive} = \text{drive frequency}$

Response (motion) will be SHM at ω_{drive} and not ω_0 !

- measure response amplitude x_{max} vs. ω_{drive} for fixed drive amplitude:



for many systems:

- resonance width

$$\Delta\omega = \frac{1}{\tau}$$

energy decay time

- define quality factor:

$$Q \equiv \frac{\omega_0}{\Delta\omega} = \omega_0 \tau = \frac{2\pi\tau}{T_0}$$

$$= 2\pi \left(\begin{array}{l} \# \text{ of oscillations at} \\ \omega_0 \text{ in time interval } \Delta t = \tau \end{array} \right)$$

- amplitude at resonance $\propto Q \propto \tau$

\propto "time the system remembers energy
force amplitude input"

- specific model:

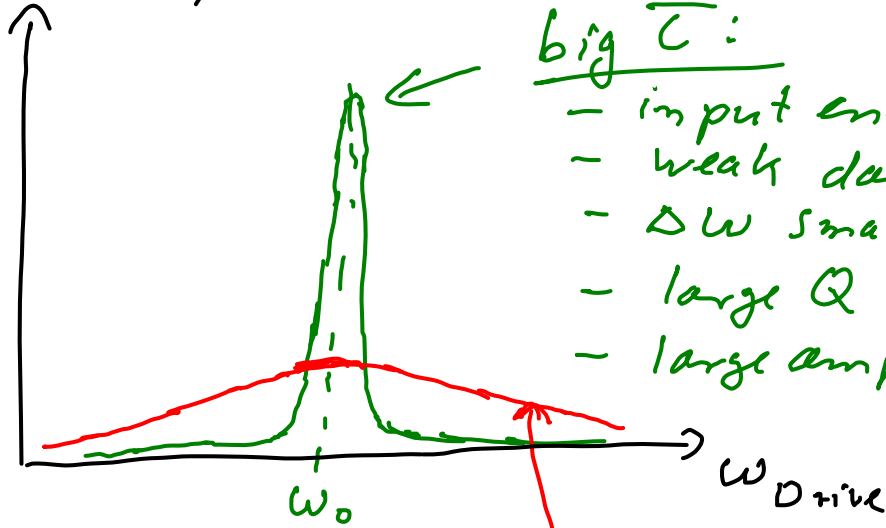
$F_{\max}/m \leftarrow$ mass of obj.

$$x_{\max}(\omega_{\text{drive}}) = \frac{F_{\max}/m}{\sqrt{(\omega_{\text{drive}}^2 - \omega_0^2)^2 + \left(\frac{\omega_{\text{drive}}}{\tau}\right)^2}}$$

\Rightarrow max. amplitude at

$$\omega_{\text{drive}} = \sqrt{\omega_0^2 - \frac{1}{2\tau^2}} \approx \omega_0 \text{ for small damping, i.e. large } \tau$$

$X_m(\omega_{\text{Drive}})$



big τ :

- input energy decays slowly
- weak damping
- $\Delta\omega$ small
- large Q
- large amplitude at resonance

Small τ :

- input energy decays fast
- strong damping
- $\Delta\omega$ large
- small Q
- small amplitude at resonance

Resonance and Driven Oscillations in Animal Movement

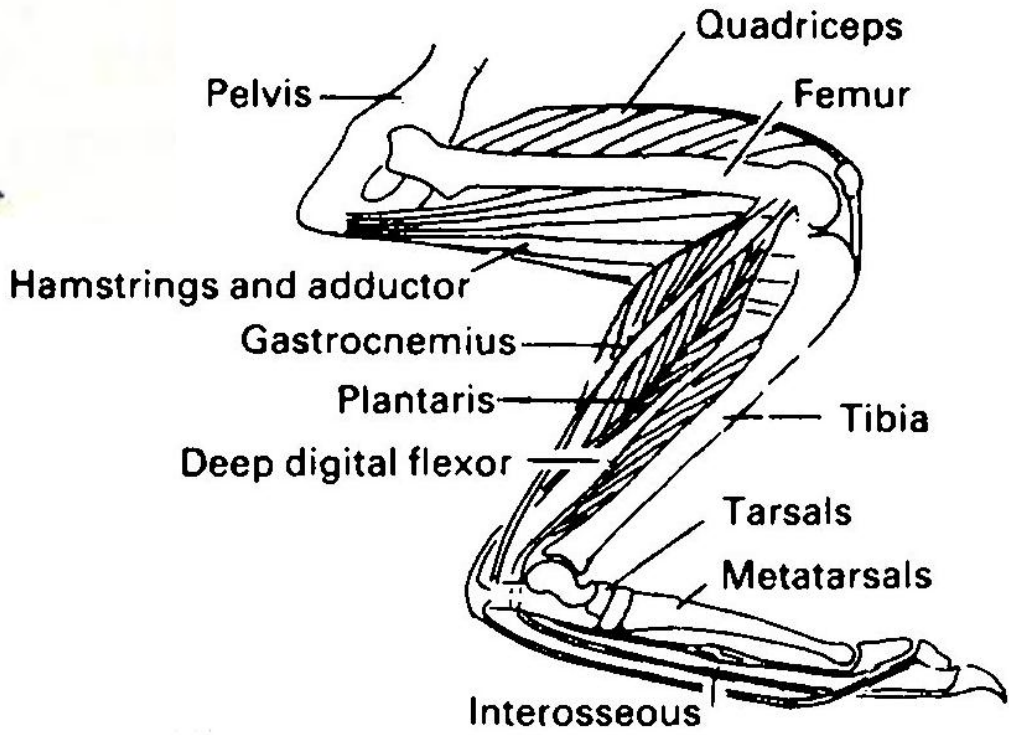
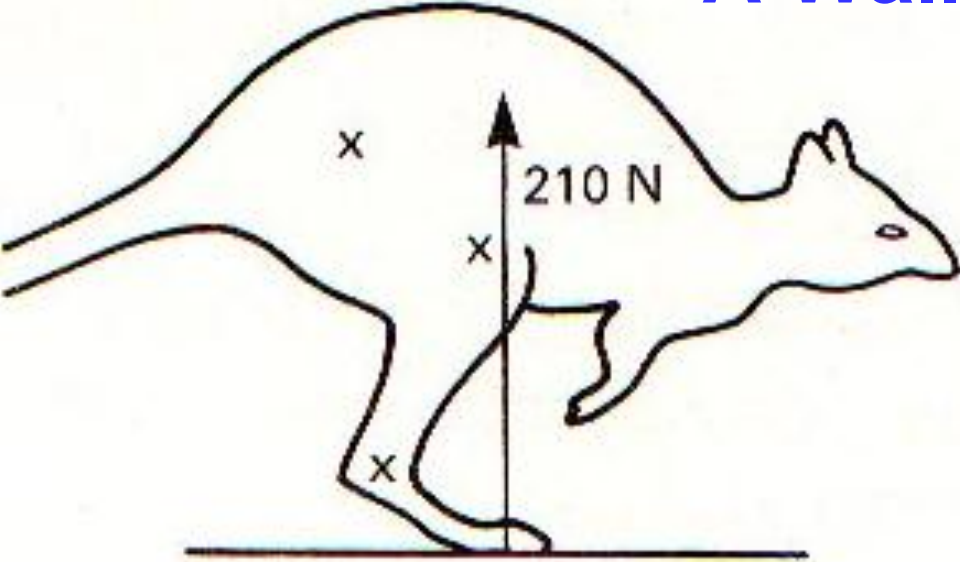
Problem: Nature didn't invent the wheel

- Animal motion requires that parts of the animal speed up and slow down, with a corresponding variation in kinetic energy.
- Doing this with muscles alone would waste too much energy.

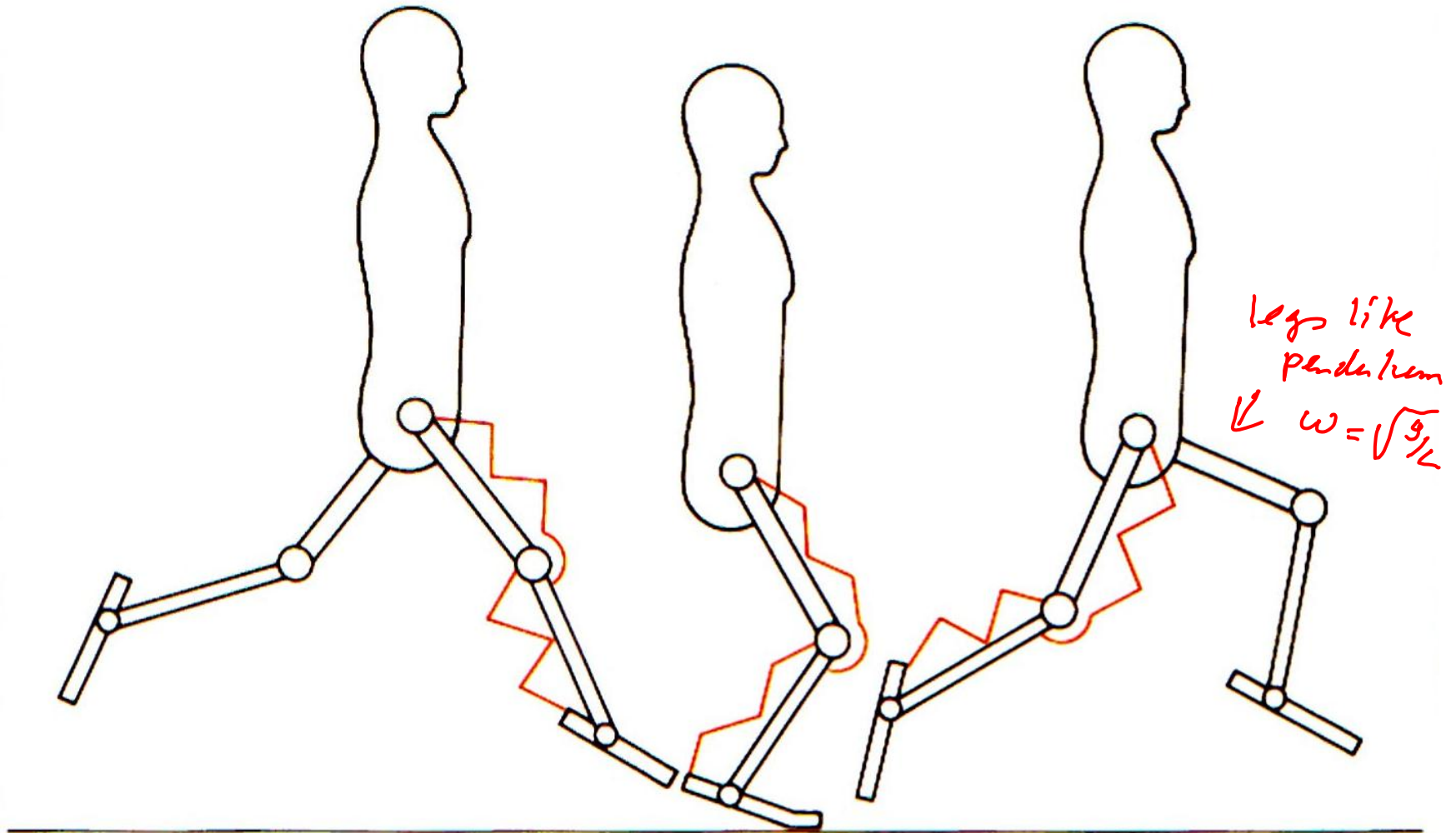
Solution:

- Use **springs** (**tendons, ligaments, muscles**) to convert **kinetic energy** to and from **elastic potential energy**.
- **Drive the system near its resonant frequency to get the maximum motion for a given energy input.**

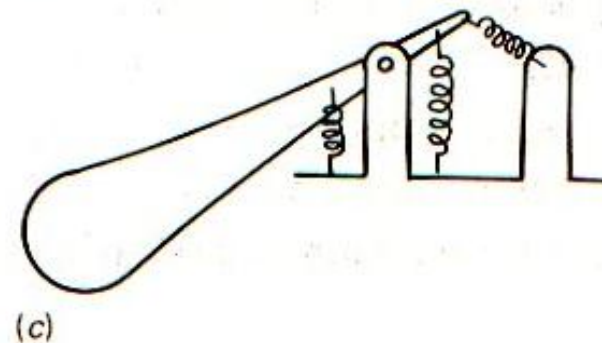
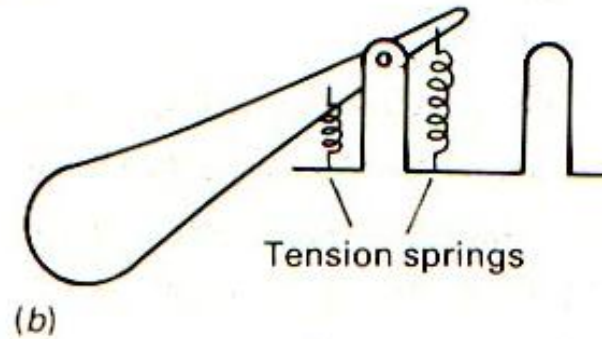
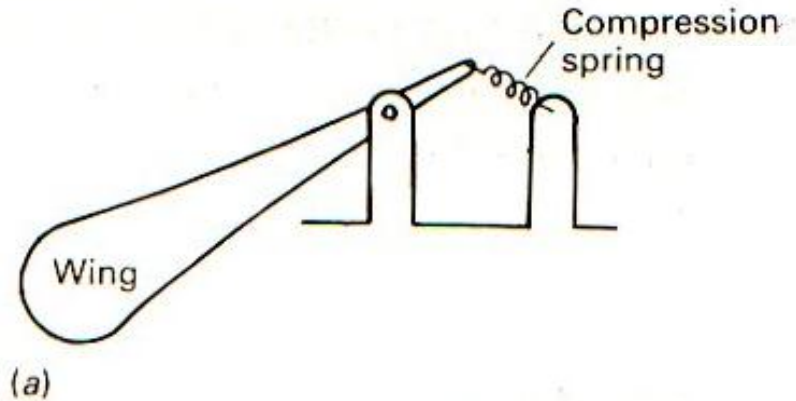
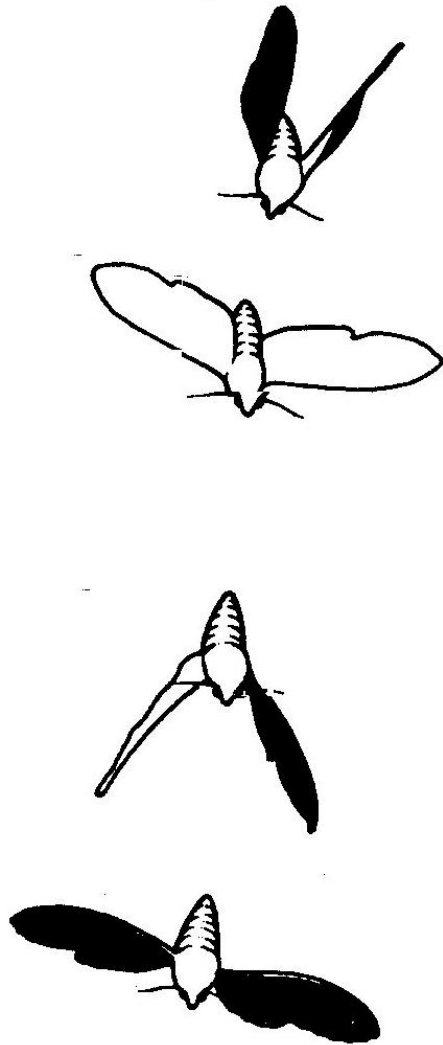
A Wallaby Hopping:



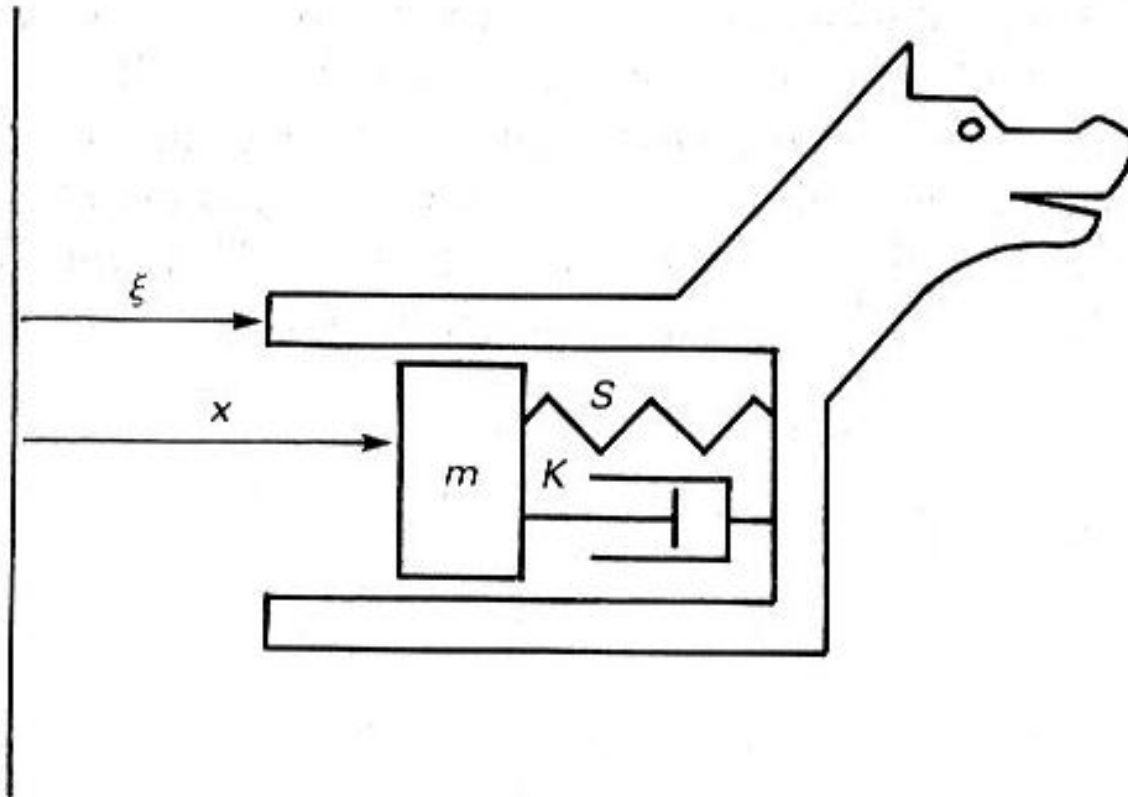
A Human Running:



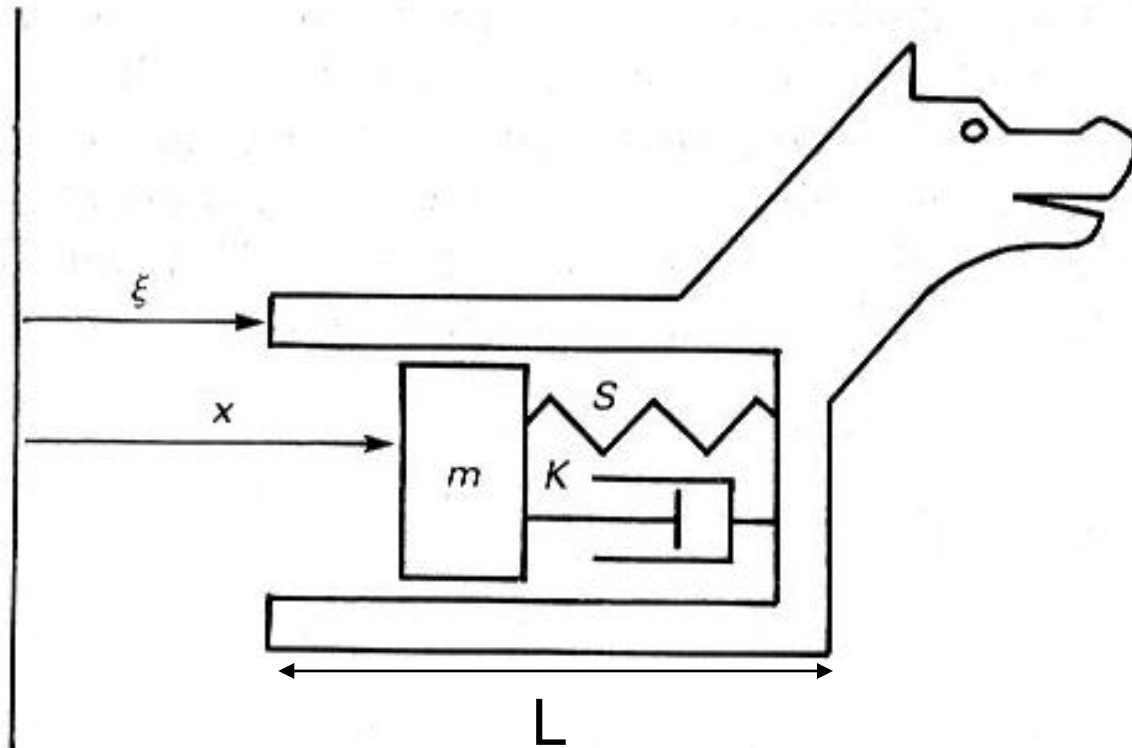
An Insect Flying:



A Dog Panting:



A Dog Panting:



$$\begin{aligned} m &\propto L^3, & k &\propto A/L \propto L \\ \therefore T = 2\pi(m/k)^{1/2} &\Rightarrow T \propto L \propto \text{size of dog} \end{aligned}$$